

IYPT 2010

№13 SHRIEKING ROD

GEORGIAN Team

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A metal rod is held between two fingers and hit. Investigate how the sound produced depends on the position of holding and hitting the rod?

Presentation Plan

a) Wave types in rod

- ❖ Longitudinal Waves
- ❖ Bending Waves
- ❖ Torsion waves

b) Longitudinal waves

- ❖ Wave equation
- ❖ Solution of Wave equation
- ❖ Frequency modes ; Node positions

c) Experiments with Longitudinal waves

d) Bending waves

- ❖ Wave equation
- ❖ Solution of Wave equation
- ❖ Frequency modes ; Node positions

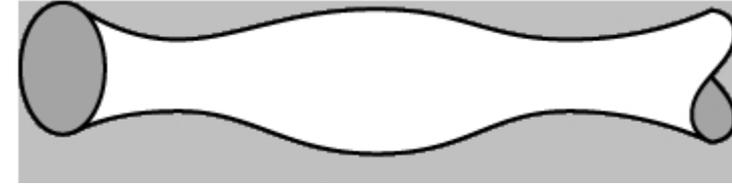
e) Experiments with Bending waves

f) Conclusion

Wave types in rod

(A) Quasi-longitudinal **Compression waves** in a thin rod

Quasi longitudinal due to transverse strains— as rod stretches, it grows thinner



(B) **Bending waves** in a thin rod

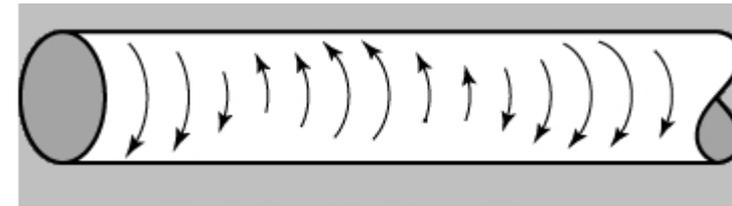
Bending waves involve both compression and shear strains. Their velocity depends on frequency - they are **DISPERSIVE**



(C) Transverse **Torsion waves** in a thin rod

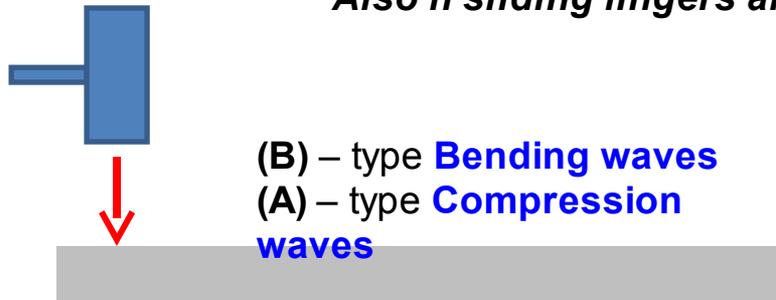
A lateral displacement χ which varies with x gives rise to a **shear strain**.

In a thin rod torsion shear waves travel at a speed which is always little less than longitudinal wave speed.

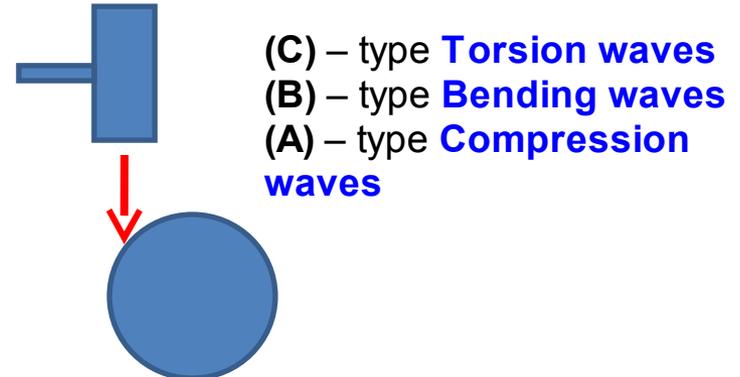


(A) – type **Compression waves** (*in ideal case*)

Also if sliding fingers along the rod



(B) – type **Bending waves**
(A) – type **Compression waves**

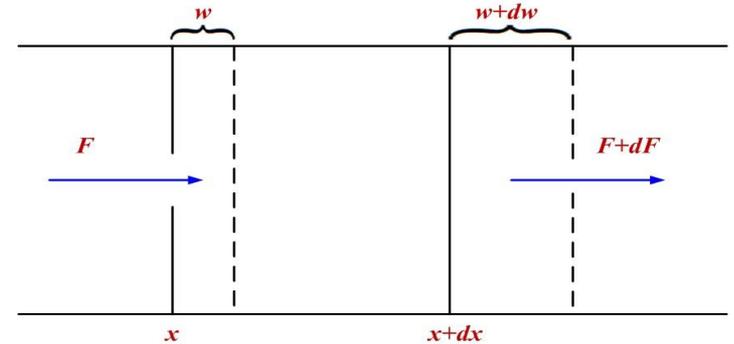


(C) – type **Torsion waves**
(B) – type **Bending waves**
(A) – type **Compression waves**

Longitudinal Waves

Wave equation

- Short segment of length dx
- Cross-section area S
- Force $F(x)$
- The plane at x moves a distance w to the right
- The **Stress** F/S
- The **Strain** (change in length per unit of original length) $\partial w / \partial x$
- The Young's module E



The **Hooke's law**:

$$\frac{F}{S} = E \frac{\partial w}{\partial x} \quad (1)$$

The net force:

$$dF_{net} = F(x + dx) - F(x) = \frac{\partial F}{\partial x} dx = SE \frac{\partial^2 w}{\partial x^2} dx$$

The **Newton's second law**:

$$dF_{net} = a \cdot dm = \rho S dx \frac{\partial^2 w}{\partial t^2} = SE \frac{\partial^2 w}{\partial x^2} dx$$

or

$$\frac{\partial^2 w}{\partial t^2} = \frac{E}{\rho} \frac{\partial^2 w}{\partial x^2} = c_L^2 \frac{\partial^2 w}{\partial x^2} \quad (2)$$

- one-dimensional wave equation for waves with a **velocity**

$$c_L = \sqrt{E/\rho} \quad (3)$$

Solution of Wave equation

Let us search the **harmonic standing wave modes** in the form: $w = f(x)\text{Cos}(\omega t - \varphi)$

Substituting in (2) gives:
$$\frac{d^2 f(x)}{dx^2} = -\frac{\omega^2}{c_L^2} f(x)$$

This yields:
$$f(x) = A\text{Cos}(kx) + B\text{sin}(kx)$$

Where the wave number
$$k = \omega/c_L = 2\pi/\lambda \quad . \quad (4)$$

Boundary conditions for both ($x=0$; $x=l$) ends free: $F = 0 \rightarrow \frac{\partial w}{\partial x} = 0$

This gives : $B = 0$; $\text{Sin}(kl) = 0 \rightarrow k = \frac{\pi n}{l} \rightarrow \omega = \frac{\pi n c_L}{l}$.

i.e.

Frequency modes:
$$v_n = \frac{\omega}{2\pi} = n \frac{c_L}{2l} \quad ; n = 1, 2, 3, \dots \quad (5)$$

Wave lengths:
$$\lambda_n = \frac{c_L}{v_n} = \frac{2l}{n} \quad ; n = 1, 2, 3, \dots \quad (6)$$

Wave solution:
$$w = A \text{Cos}\left(\frac{\pi n}{l} x\right) \cdot \text{Cos}\left(\frac{\pi n c_L}{l} t - \varphi\right) \quad ; \quad c_L = \sqrt{E/\rho}$$

Standing wave modes

The First Harmonic:

Wave length: $\lambda_1 = 2l$

Frequency: $\nu_1 = \frac{\sqrt{E/\rho}}{2l}$

$C_L = \sqrt{E/\rho}$

Nod positions: $l/2$

The Second Harmonic:

Wave length: $\lambda_2 = l$

Frequency: $\nu_2 = \frac{\sqrt{E/\rho}}{l} = 2\nu_1$

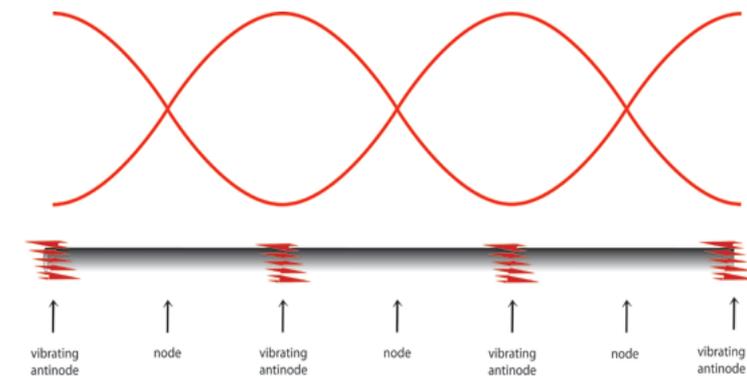
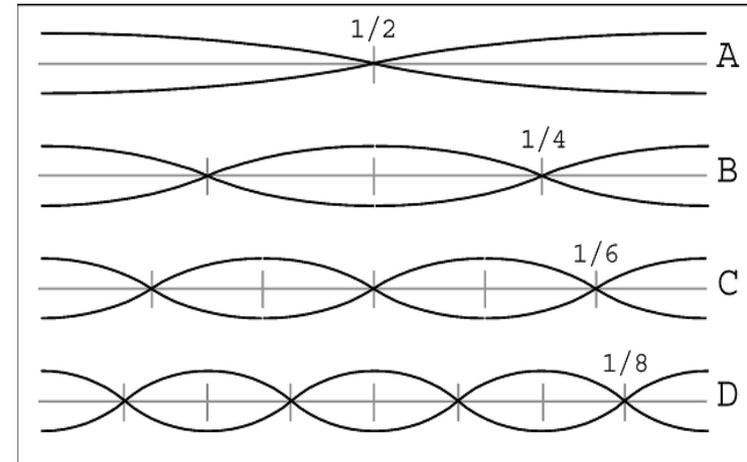
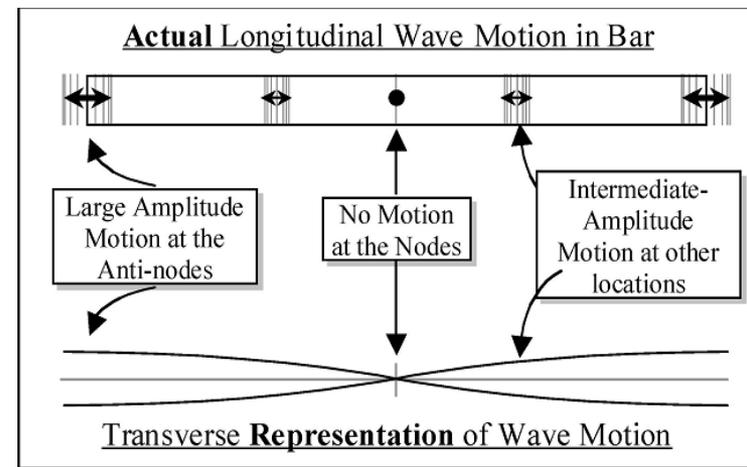
Nod positions: $l/4 ; 3l/4$

The Third Harmonic:

Wave length: $\lambda_3 = \frac{2l}{3}$

Frequency: $\nu_3 = \frac{3\sqrt{E/\rho}}{2l} = 3\nu_1$

Nod positions: $l/6 ; l/2 ; 5l/6$



The modal frequencies, wavelengths, and the locations of nodes and anti-nodes for the first nine harmonics associated with a vibrating rod of length, L

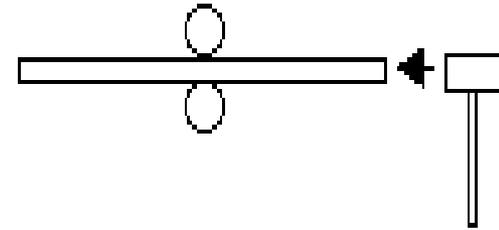
Harmonic Mode #, n	Frequency f (Hz)	Wavelength λ (m)	Node Locations, (m)	Anti-Node Locations, (m)
1	f_1	$\lambda_1 = 2L$	0	$\pm^{1/2}L$
2	$f_2 = 2f_1$	$\lambda_2 = \frac{1}{2}\lambda_1 = L$	$\pm^{1/4}L$	$0, \pm^{2/4}L = \pm^{1/2}L$
3	$f_3 = 3f_1$	$\lambda_3 = \frac{1}{3}\lambda_1 = \frac{2}{3}L$	$0, \pm^{2/6}L = \pm^{1/3}L$	$\pm^{1/6}L, \pm^{3/6}L = \pm^{1/2}L$
4	$f_4 = 4f_1$	$\lambda_4 = \frac{1}{4}\lambda_1 = \frac{1}{2}L$	$\pm^{1/8}L, \pm^{3/8}L$	$0, \pm^{2/8}L = \pm^{1/4}L, \pm^{4/8}L = \pm^{1/2}L$
5	$f_5 = 5f_1$	$\lambda_5 = \frac{1}{5}\lambda_1 = \frac{2}{5}L$	$0, \pm^{2/10}L = \pm^{1/5}L, \pm^{4/10}L = \pm^{2/5}L$	$\pm^{1/10}L, \pm^{3/10}L, \pm^{5/10}L = \pm^{1/2}L$
6	$f_6 = 6f_1$	$\lambda_6 = \frac{1}{6}\lambda_1 = \frac{1}{3}L$	$\pm^{1/12}L, \pm^{3/12}L = \pm^{1/4}L, \pm^{5/12}L$	$0, \pm^{2/12}L = \pm^{1/6}L, \pm^{4/12}L = \pm^{1/3}L, \pm^{6/12}L = \pm^{1/2}L$
7	$f_7 = 7f_1$	$\lambda_7 = \frac{1}{7}\lambda_1 = \frac{2}{7}L$	$0, \pm^{2/14}L = \pm^{1/7}L, \pm^{4/14}L = \pm^{2/7}L, \pm^{6/14}L = \pm^{3/7}L$	$\pm^{1/14}L, \pm^{3/14}L, \pm^{5/14}L, \pm^{7/14}L = \pm^{1/2}L$
8	$f_8 = 8f_1$	$\lambda_8 = \frac{1}{8}\lambda_1 = \frac{1}{4}L$	$\pm^{1/16}L, \pm^{3/16}L, \pm^{5/16}L, \pm^{7/16}L$	$0, \pm^{2/16}L = \pm^{1/8}L, \pm^{4/16}L = \pm^{1/4}L, \pm^{6/16}L = \pm^{3/8}L$
9	$f_9 = 9f_1$	$\lambda_9 = \frac{1}{9}\lambda_1 = \frac{2}{9}L$	$0, \pm^{2/18}L = \pm^{1/9}L, \pm^{4/18}L = \pm^{2/9}L, \pm^{6/18}L = \pm^{1/3}L, \pm^{8/18}L = \pm^{4/9}L$	$\pm^{1/18}L, \pm^{3/18}L, \pm^{5/18}L, \pm^{7/18}L, \pm^{9/18}L = \pm^{1/2}L$

All modes ($n = 0, 1, 2, 3, 4, 5, \dots$) of vibration of the rod all have the same longitudinal speed of propagation of sound in the rod $C_L = \sqrt{E/\rho} = v_i \lambda_i$

Experiments with Longitudinal Waves

To make rod sound **clearer and louder**, we shall:

3. Hit the rod **as fast as possible**
4. Hit it **not very hard**, because harmonicity of waves will be violated.
5. Hit the rod with thing, that **doesn't produce good sound** (e.g. Ebonite Rod), not to interrupt Main one.
6. Try to hit it **vertically or horizontally**, and **not intermediate**.



- Also **sliding of hand** down the length of the rod.
- Addition of **Rosin** makes **fingers more sticky**.
- The **pitch** of the sound can be **varied** by changing **holding places of the Rod** or by changing the **length of the rod** itself.

Ringling The Rod



Experiment

Experiments with Longitudinal Waves

1. **Aluminium rod** ($\rho_{AL} = 2,7 \cdot 10^3 \text{ kg/m}^3$; $E_{AL} = 70 \times 10^9 \text{ N/m}^2$; $C_{L(AL)} = 5082.4 \text{ m/s}$)

Length of the rod 1: $l = 1,2 \text{ m}$ $v_1 = \frac{\sqrt{E/\rho}}{2l}$ $C_L = \sqrt{E/\rho}$ $\lambda_n = \frac{2l}{n}$ $v_n = n \frac{C_L}{2l}$

The first harmonic: $\lambda_1 = 2.4 \text{ m}$; $v_1 = 2117 \text{ Hz}$; nodes: 0.6 m

The second harmonic: $\lambda_2 = 1.2 \text{ m}$; $v_2 = 4234 \text{ Hz}$; nodes: 0.3 m , 0.9 m

The third harmonic: $\lambda_3 = 0.8 \text{ m}$; $v_3 = 6351 \text{ Hz}$; nodes: 0.2 m , 0.6 m , 1.0 m

If hold the rod in the place where nodes of several modes are placed all these modes will occur.

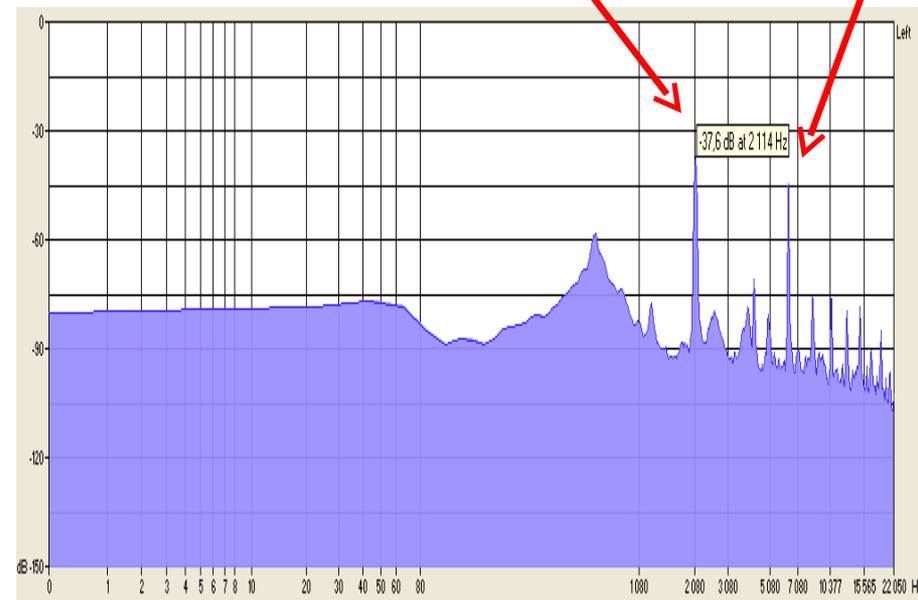
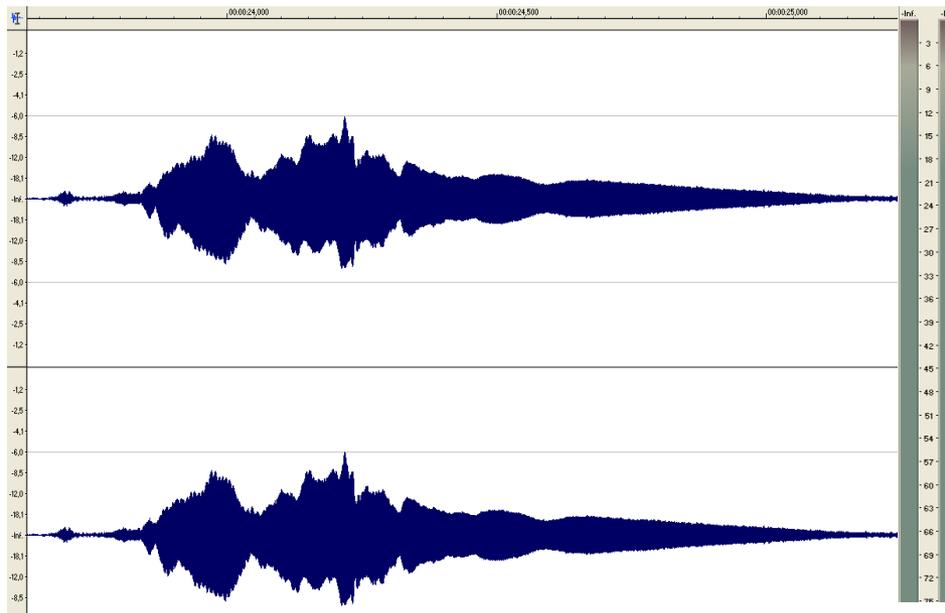
Touching the rod at the ends will stop the sound .



????? ??????????

holding in center v_1

v_3



Experiments with Longitudinal Waves

2. **Aluminium rod** ($\rho_{AL} = 2,7 \cdot 10^3 \text{ kg/m}^3$; $E_{AL} = 70 \times 10^9 \text{ N/m}^2$; $C_{L(AL)} = 5082.4 \text{ m/s}$)

Length of the rod 2: $l = 0.75 \text{ m}$ $v_1 = \frac{\sqrt{E/\rho}}{2l}$ $C_L = \sqrt{E/\rho}$ $\lambda_n = \frac{2l}{n}$ $v_n = n \frac{C_L}{2l}$

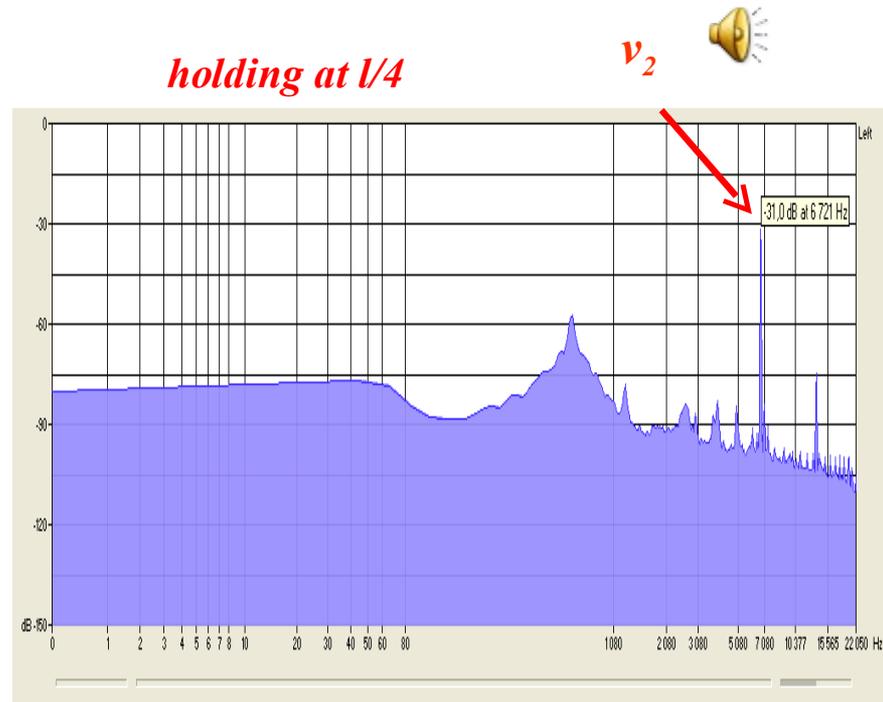
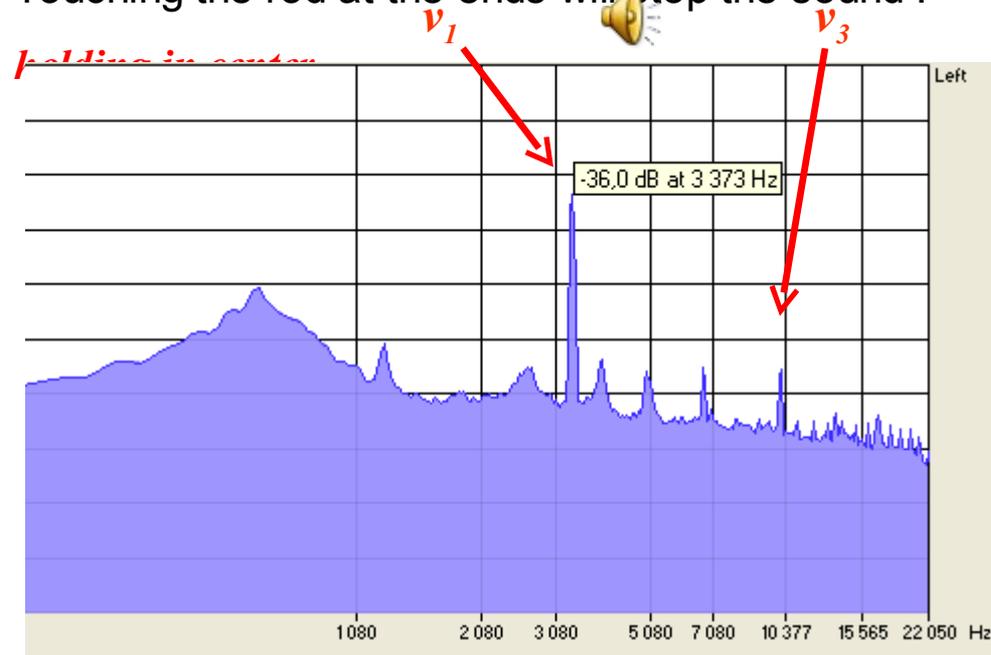
The first harmonic: $\lambda_1 = 1.5 \text{ m}$; $v_1 = 3388 \text{ Hz}$; nodes: 0.37 m

The second harmonic: $\lambda_2 = 0.75 \text{ m}$; $v_2 = 6776 \text{ Hz}$; nodes: 0.19 m , 0.56 m

The third harmonic: $\lambda_3 = 0.5 \text{ m}$; $v_3 = 10164 \text{ Hz}$; nodes: 0.12 m , 0.37 m , 0.63 m

If hold the rod in the place where nodes of several modes are placed all these modes will occur.

Touching the rod at the ends will stop the sound .



Experiments with Longitudinal Waves

3. **BRASS rod** ($\rho_{Br} = 8,5 \cdot 10^3 \text{ kg/m}^3$; $E_{Br} = 95 \times 10^9 \text{ N/m}^2$; $C_{L(Br)} = 3480 \text{ m/s}$)

Length of the rod 3: $l = 0.4 \text{ m}$ $v_1 = \frac{\sqrt{E/\rho}}{2l}$ $C_L = \sqrt{E/\rho}$ $\lambda_n = \frac{2l}{n}$ $v_n = n \frac{C_L}{2l}$

The first harmonic: $\lambda_1 = 0.8 \text{ m}$; $v_1 = 4350 \text{ Hz}$; nodes: 0.2 m

The second harmonic: $\lambda_2 = 0.4 \text{ m}$; $v_2 = 8700 \text{ Hz}$; nodes: 0.1 m , 0.3 m

The third harmonic: $\lambda_3 = 0.27 \text{ m}$; $v_3 = 13\ 050 \text{ Hz}$; nodes: 0.07 m , 0.2 m , 0.33 m

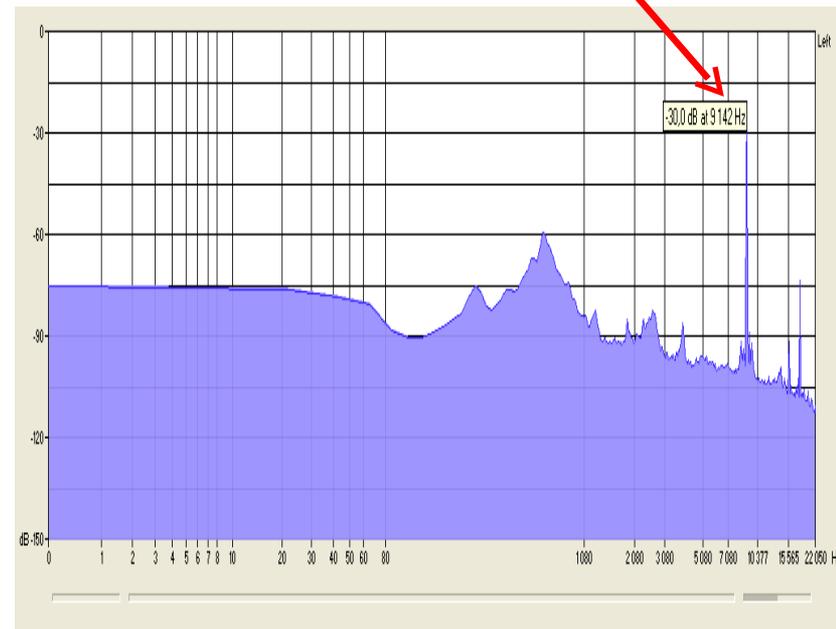
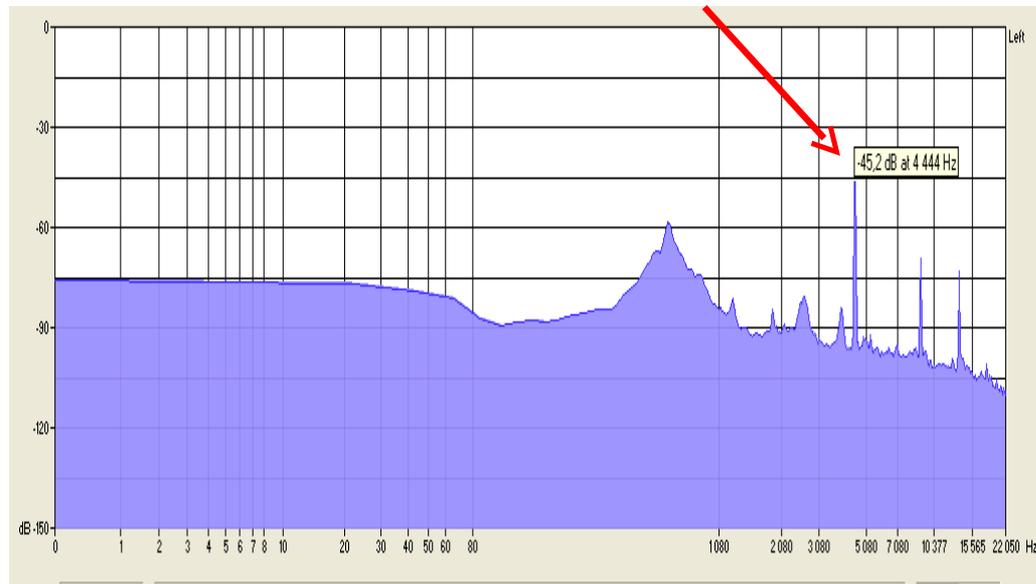
If hold the rod in the place where nodes of several modes are placed all these modes will occur.

Touching the rod at the ends will stop the sound .
holding in center

v_1 

holding at 1/4

v_2 



Experiments with Longitudinal Waves

4. **STEEL rod** ($\rho_{St} = 7,8 \cdot 10^3 \text{ kg/m}^3$; $E_{St} = 200 \times 10^9 \text{ N/m}^2$; $C_{L(St)} = 5150 \text{ m/s}$)

Length of the **rod 4**: $l = 0.6 \text{ m}$ $v_1 = \frac{\sqrt{E/\rho}}{2l}$ $C_L = \sqrt{E/\rho}$ $\lambda_n = \frac{2l}{n}$ $v_n = n \frac{C_L}{2l}$

The first harmonic: $\lambda_1 = 1.2 \text{ m}$; $v_1 = 4291 \text{ Hz}$; **nodes: 0.3 m**

The second harmonic: $\lambda_2 = 0.6 \text{ m}$; $v_2 = 8582 \text{ Hz}$; **nodes: 0.15 m , 0.45 m**

The third harmonic: $\lambda_3 = 0.4 \text{ m}$; $v_3 = 12\,873 \text{ Hz}$; **nodes: 0.1 m , 0.3 m , 0.5 m**

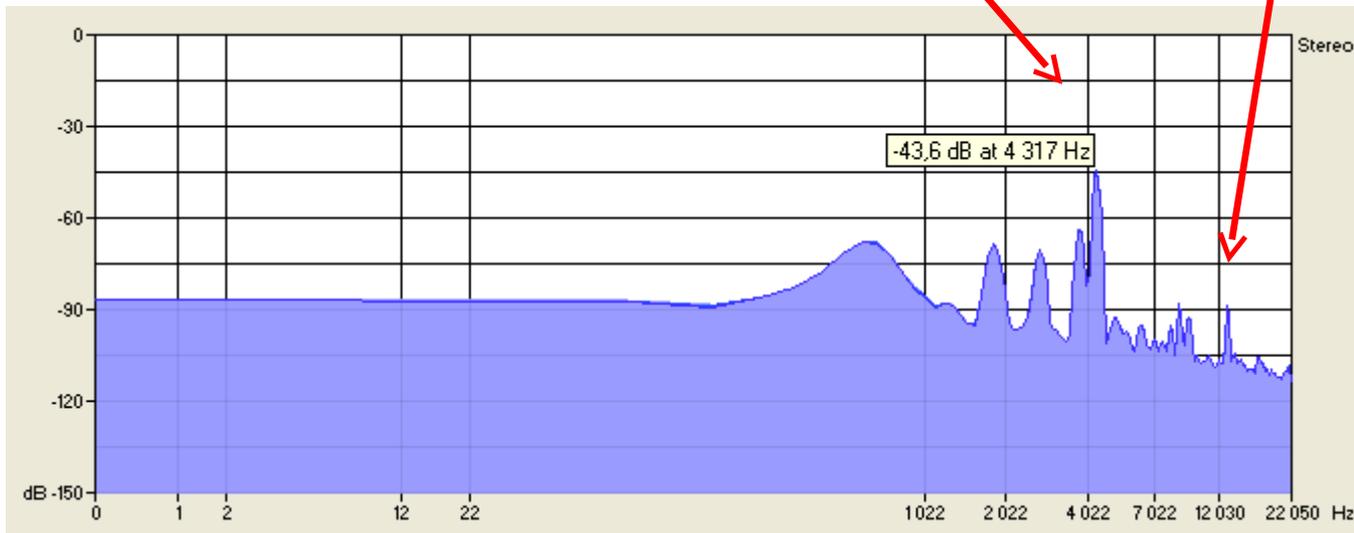
If hold the rod in the place where nodes of several modes are placed all these modes will occur.

Touching the rod at the ends will stop the sound

holding in center

v_1

v_3



BENDING Waves

Wave equation

Euler-Bernoulli beam theory equation of motion

$$\frac{\partial^2 y}{\partial t^2} = - \frac{EK^2}{\rho} \cdot \frac{\partial^4 y}{\partial x^4}$$

Where y is displacement normal to rod axis.

"Radius of Gyration":
$$K^2 = \frac{1}{S} \int z^2 dS$$

z is distance from central axis of rod.

Here shear deformations and rotary inertia are neglected.

Harmonic solutions are of the following form:

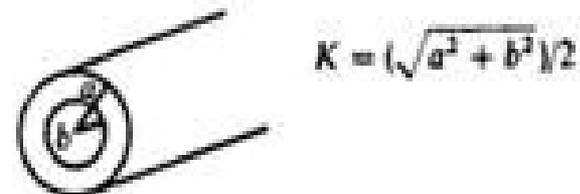
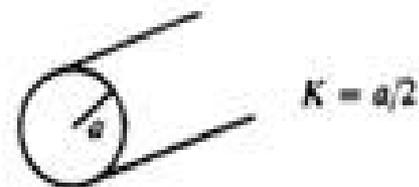
$$y(x, t) \equiv Y(x)e^{i\omega t}$$

$$Y(x) = \cos(\omega t + \varphi) [A \cosh(kx) + B \sinh(kx) + C \cos(kx) + D \sin(kx)]$$

$k = \omega/v$ is the wave (propagation) number.

Velocity is dependent on frequency. So here is dispersion.

$$v = \sqrt{2\pi\nu Kc_L}$$



$$K = \frac{\sqrt{(\text{inner radius})^2 + (\text{outer radius})^2}}{2}$$

Frequency Modes of Standing Bending Waves



Frequency modes depend on the end conditions

For our task we consider FREE end - no torque and no shearing force

They give restrictions on standing wave frequencies:

$$\frac{\omega l}{2v} = \frac{\pi}{4} \cdot [3.011 ; 5 ; 7 ; \dots ; (2n + 1) \dots]$$

$$v = \sqrt{2\pi\nu K c_L}$$

$$\nu_n = \frac{\omega_n}{2\pi} = \frac{\pi K}{8l^2} \sqrt{\frac{E}{\rho}} \cdot [3.011^2 ; 5^2 ; 7^2 ; \dots ; (2n + 1)^2 \dots]$$

And wave lengths:

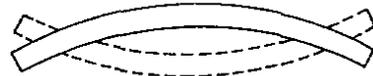
$$\lambda_n = \frac{4l}{[3.011 ; 5 ; 7 ; \dots ; (2n + 1) \dots]}$$

Table I. Characteristics of transverse vibrations in a bar with free ends.

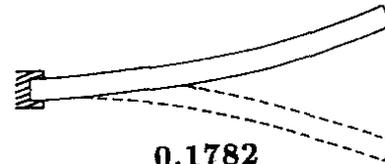
Frequency (Hz)	Wavelength (m)	Nodal positions (m from end of 1-m bar)
$f_1 = 3.5607 (K/L^2)\sqrt{E/\rho}$	1.330L	0.224, 0.776
$2.756 f_1$	0.800L	0.132, 0.500, 0.868
$5.404 f_1$	0.572L	0.094, 0.356, 0.644, 0.906
$8.933 f_1$	0.445L	0.073, 0.277, 0.500, 0.723, 0.927

To obtain actual frequencies multiply

$$\frac{\pi K}{l^2} \sqrt{\frac{E}{\rho}}$$



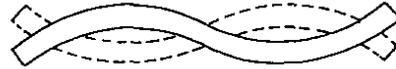
1.133



0.1782



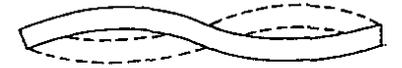
0.50



3.125



1.116



2.00



6.125



3.125



4.50



10.125



6.125



8.00

(a)

(b)

(c)

Note: Frequencies for bending waves are sufficiently lower than for the longitudinal waves due to extra coefficient $\frac{K}{l}$.

Higher modes are vanishing very rapidly.

Waves are damped for supported or clamped ends

Experiments with BENDING Waves

1. **Aluminium rod** ($\rho_{AL} = 2,7 \cdot 10^3 \text{ kg/m}^3$; $E_{AL} = 70 \times 10^9 \text{ N/m}^2$; $C_{L(AL)} = 5082.4 \text{ m/s}$)

FREE Ends

$$v_n = \sqrt{2\pi v_n K C_L} \quad ; \quad K = \frac{1}{2} \sqrt{a^2 + b^2} \quad ; \quad v_n = \frac{\pi K}{8l^2} \sqrt{\frac{E}{\rho}} \cdot [3.011^2; 5^2; 7^2; \dots; (2n+1)^2 \dots]$$

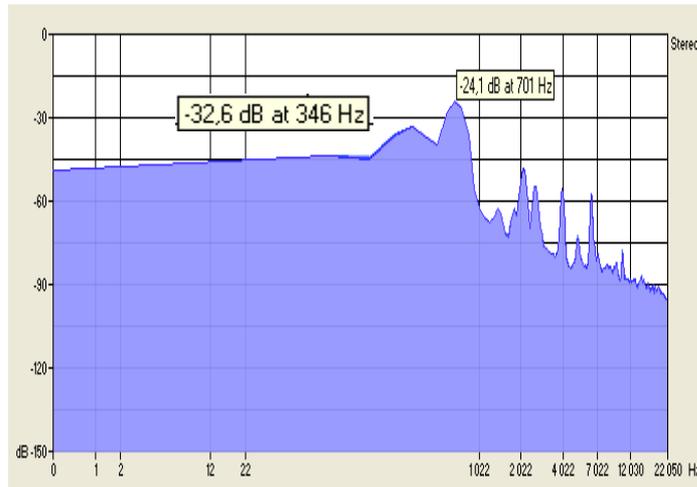
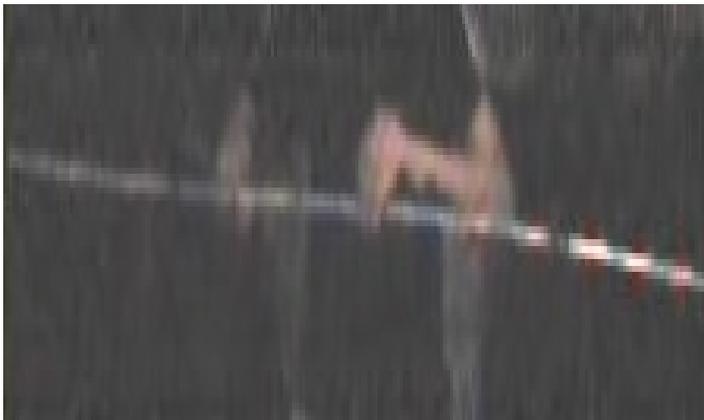
$$\lambda_n = \frac{4l}{[3.011; 5; 7; \dots; (2n+1) \dots]}$$

Length of the **rod 1**: $l = 1,2\text{m}$; Radii: $a = 0.005\text{m}$, $b = 0.004\text{m}$; $K = 0.0032\text{m}$

The first harmonic: $\lambda_{B1} = 1,596\text{m}$; $v_1 = 40 \text{ Hz}$; nodes: 0.27m , $0,93\text{m}$

The second harmonic: $\lambda_{B2} = 0,96\text{m}$; $v_2 = 110 \text{ Hz}$; nodes: 0.16m , 0.6m , 1.04m

The third harmonic: $\lambda_{B3} = 0.69\text{m}$; $v_3 = 215 \text{ Hz}$; nodes: 0.11m , 0.43m , 0.77m



holding in center



Experiments with BENDING Waves

4. **STEEL rod** ($\rho_{St} = 7,8 \cdot 10^3 \text{ kg/m}^3$; $E_{St} = 200 \times 10^9 \text{ N/m}^2$; $C_{L(St)} = 5150 \text{ m/s}$)

FREE Ends

$$v_n = \sqrt{2\pi v_n K c_L} \quad ; \quad K = a/2 \quad ; \quad v_n = \frac{\pi K}{8l^2} \sqrt{\frac{E}{\rho}} \cdot [3.011^2; 5^2; 7^2; \dots; (2n+1)^2 \dots]$$

$$\lambda_n = \frac{4l}{[3.011; 5; 7; \dots; (2n+1) \dots]}$$

Length of the **rod 4**: $l = 0.6 \text{ m}$; **Radius** $a = 0.007 \text{ m}$; $K = 0.0035 \text{ m}$

The first harmonic: $\lambda_{B1} = 0.8 \text{ m}$; $v_1 = 190 \text{ Hz}$; **nodes**: 0.13 m , 0.47 m

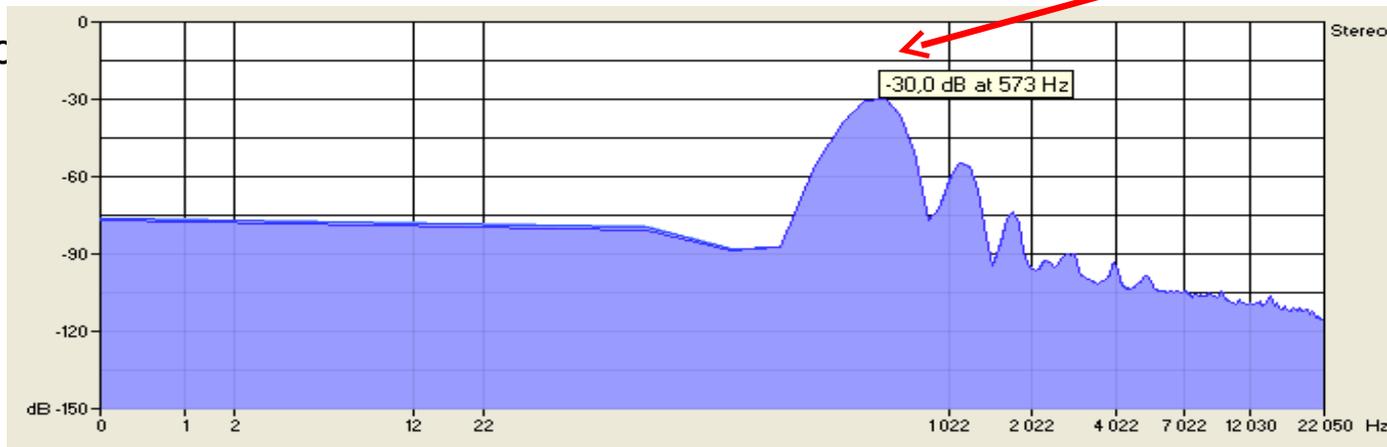
The second harmonic: $\lambda_{B2} = 0.48 \text{ m}$; $v_2 = 525 \text{ Hz}$; **nodes**: 0.08 m , 0.3 m , 0.52 m

The third harmonic: $\lambda_{B3} = 0.34 \text{ m}$; $v_3 = 1029 \text{ Hz}$; **nodes**: 0.056 m , 0.21 m , 0.39 m , 0.544 m

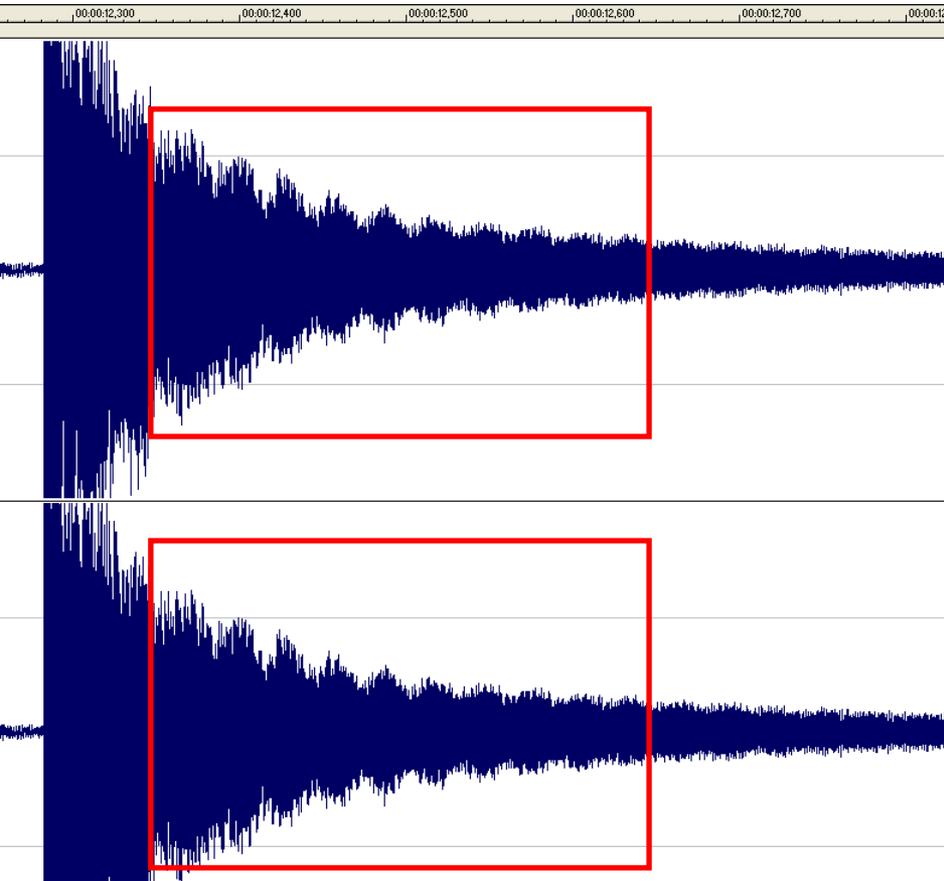
If hold the rod in the place where nodes of several modes are placed ~~at~~ all these modes will occur.

holding at 0.3m

Touch



Beats



Sound in Experiments **sometimes** became **stronger, sometimes weaker.**

This was **because of Beats.**

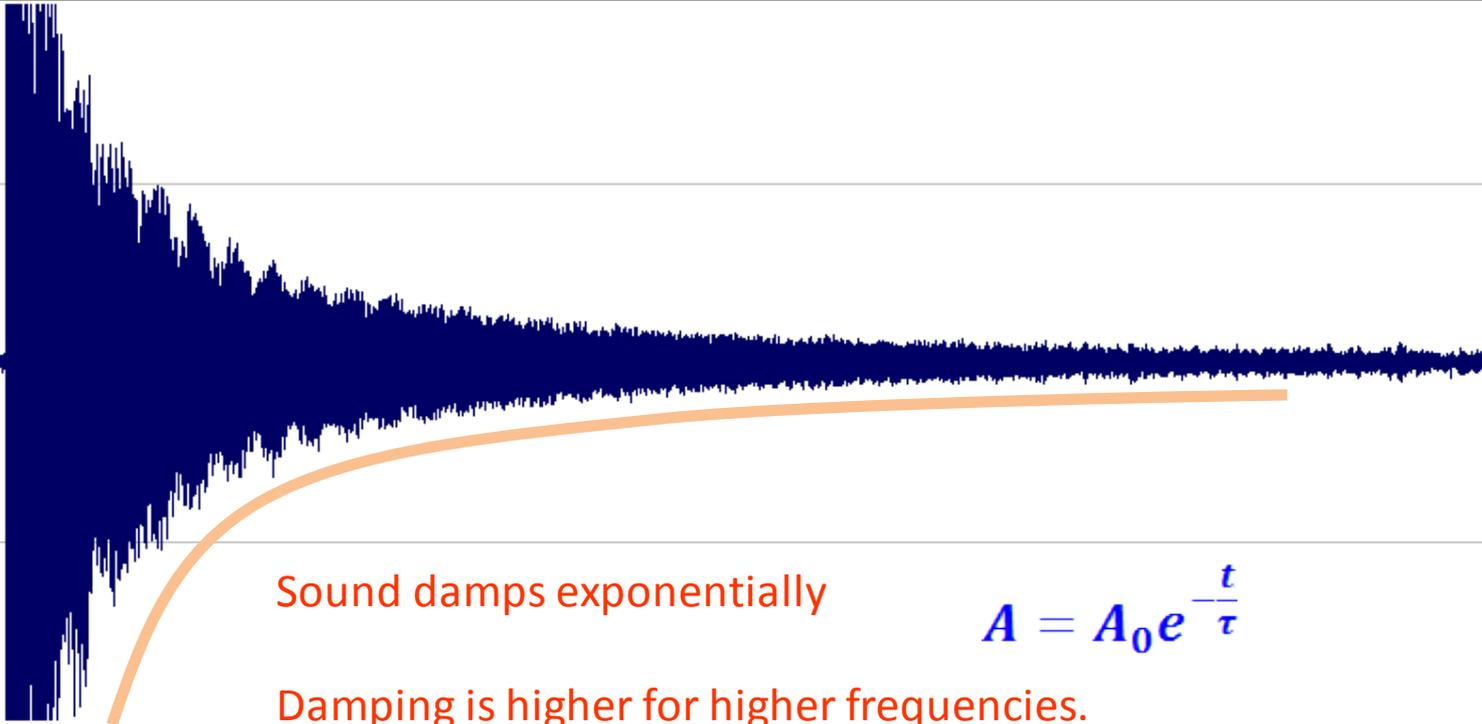
A **beat** is an **interference** between **two sounds** of **slightly different frequencies.**

Our experimental Beats

Sound Damping

00:00:12,500

00:00:13,000



Sound damps exponentially

$$A = A_0 e^{-\frac{t}{\tau}}$$

Damping is higher for higher frequencies.

We saw it comparing damping for transverse and longitudinal waves.

- Internal Damping

The "decay time"

$$\tau_{int} \sim \frac{1}{\pi v}$$

- Air damping.

The "decay time"

$$\tau_a \sim \rho r / \sqrt{v}$$

- Transfer of energy to other systems (e.g. supports)

$$\tau_{sup} \sim \frac{1}{lGv^2} \quad (G- \text{"energy conductance"})$$

Conclusion

- There are different types of waves in the rod
- The type of wave depends on how do we hit the rod.
- The frequency of standing wave depends on where we hold the rod.
- The frequencies of bending waves are lower than of compression waves
- Damping of the waves depends on frequency

Thank you for your attention!